

Quadrant Analysis of Turbulence in a Rectangular Cavity with Large Aspect Ratios

Dawei Guan¹; Prakash Agarwal²; and Yee-Meng Chiew, M.ASCE³

Abstract: By using particle image velocimetry techniques, this paper presents new insights on the turbulence structure and time-averaged flows in a rectangular cavity with large aspect ratios. The three cavity aspect ratios (L/D , where L = cavity length, D = cavity depth) used in this study are 9.9, 13.0, and 18.9. The time-averaged velocity fields and Reynolds shear stress distributions within the cavity show that the flow pattern and turbulence structures are strongly affected by the cavity aspect ratio. The quadrant dynamic analysis of velocity fluctuations on the cavity shear layers and cavity downstream edge for the three cavities is conducted. Considering all of the bursting events with hole size parameter, $H = 0$ in the measured planes, sweeps are found to have the highest probability of occurrence within the zone of the shear layer, and ejections dominate almost all of the areas within the cavity and a certain depth of flow immediately above the shear layer. For the high-magnitude events ($H = 1$), quadrant dominances are almost inverted for all of the measured planes. The quadrant shear stress distributions show that the opposing events are approximately balanced for these regions on the cavity shear layers and along the cavity downstream edge. DOI: 10.1061/(ASCE)HY.1943-7900.0001480. © 2018 American Society of Civil Engineers.

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Introduction

Cavity flows have always presented great practical interests for engineers and researchers due to their various engineering applications, such as pollutant dispersion (Chang et al. 2007), flow energy exchanges (Botella and Peyret 1998), and sediment transport (Agarwal 2017; Lai et al. 2010). Extensive studies on the flow characteristics within an open cavity have been conducted in the past few decades, showing that the characteristics of cavity flows not only are related to the flow conditions such as flow compressibility, viscosity, free-stream velocity, flow depth, boundary layer thickness and turbulence levels (Chang et al. 2007; Immer et al. 2016; Rowley and Williams 2006), but are also sensitively dependent on the geometry of the cavity (Ozalp et al. 2010; Zhang and Hubert 2016). For cavities with small aspect ratios ($L/D < 5$, where L = cavity length, D = cavity depth), many studies conducted in either wind tunnels or open channels have experimentally and numerically shown that the flow within the cavity is characterized by large recirculation regions, and the associated shear layer only bridges the upstream and downstream edges of the cavity without sweeping into it (Botella and Peyret 1998; Erturk et al. 2005). For cavities with large aspect ratios ($L/D > 5$), however, the flow pattern within the cavities is significantly different from those in cavities

with small aspect ratios; the shear layer developed at the cavity leading edge sweeps into the cavity, interacting with the recirculation region (Ukeiley and Murray 2005; Zdanski et al. 2006). For cavities with infinite lengths and finite depths (L/D approximately $= \infty$), the flow patterns near the cavity upstream and downstream edges can be separately regarded as a backward-facing step flow and a forward-facing step flow, respectively, as they cannot interact with each other. The characteristics of such typical step flows can be found in Spazzini et al. (2001) and Ren and Wu (2011). In addition, cavity flows, which are time and space dependent, consist of a complex array of evolving turbulent structures with different scales. This attribute greatly increases the challenge associated with the conduct of experimental works in quantifying the time- or phase-averaged cavity shear layers. Over the past few decades, the development of the particle image velocimetry (PIV) technique has made great contributions to the understanding of turbulence associated with cavity flows. By using the PIV technique, Ashcroft and Zhang (2005) studied the vortical structures over rectangular cavities and found that the size and growth rate of coherent vortical structures within the cavity vary with the geometry of the cavity. Amador et al. (2006) showed that the highest Reynolds shear stress region occurs near the adjacent step edges based on their study on the flow patterns and turbulence structures over consecutive triangular cavities. Ozalp et al. (2010) presented the turbulence structures of cavity flows within different cavity geometries (rectangular, triangular, and semicircular) and found that the rectangular and triangular cavities produce much greater turbulence compared with semicircular cavity geometry. Immer et al. (2016) conducted an experimental study on boundary layer thickness effects on flow fields over a unit-ratio cavity and concluded that the boundary layer thickness has little effect on the mean flow streamlines but significantly affects the formation of shear layer spanning the cavity top. However, these studies are confined to turbulent cavity flows with relatively small cavity aspect ratios ($L/D < 5$); limited published experimental data hitherto can be found on the characteristics of detailed turbulence structures in a rectangular cavity with large aspect ratios ($L/D > 5$). Because the aspect ratio has a significant effect on the complex nature of

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turbulent cavity flows, it is both interesting and necessary to conduct detailed experimental studies on this topic. In the field, the bed of some streams or channels are paved with concrete blocks for the purpose of grade control, whose height/length ratios are commonly larger than 8. During the life span of these channels, some of the concrete blocks may be damaged or dislocated from their original position during floods, forming cavities with even larger aspect ratios. Therefore, the flow fields within these cavities are important for hydraulic engineers when a recalculation of the hydraulic conditions is needed.

Quadrant analysis, which is a simple but powerful turbulence data-processing technique, has been used widely in the investigation of complex turbulent shear flows (Dwivedi et al. 2010; Wallace 2016). By quantifying the magnitude and duration of turbulent velocity fluctuations in four quadrants, the spatially and temporally averaged turbulence structures of the flow can be better understood. This paper presents a detailed experimental examination of the contributions of the Reynolds shear stress in the streamwise x - z coordinate plane within three rectangular cavities with different (large) aspect ratios. Because the cavity length used in this study is large (1 m), the examination zones are only confined to the center (CS) and downstream corner sections (DS) of the cavities, corresponding to the two most interesting areas within the cavity for turbulence mixing and particle entrainment (Agarwal 2017; Lai et al. 2010). Agarwal (2017) found that entrainment of a particle from the cavity is closely related to the flow field at the center, because it dictates whether the particle will be swept upstream or downstream. Moreover, after reaching the downstream end of the cavity, actual entrainment of the particle is intimately connected to the local flow field at the edge of the cavity (Lai et al. 2010). From the viewpoint of particle entrainment from a large rectangular cavity, the flow field at these two locations is of primary importance.

Experiment and Methodology

Experimental Setup

The experiments were carried out in a 14-m-long, 0.60-m-deep, and 0.49-m-wide glass-sided recirculating flume in the Hydraulics Laboratory of Nanyang Technological University. The flume contains a 1-m-length by 0.49-m-width rectangular depth-adjustable cavity. The leading edge of the cavity is located 9 m downstream of the flume inlet. In this study, one constant cavity length, $L = 1$ m, and three cavity depths, $D = 5.3, 7.7,$ and 10.1 cm (corresponding to three aspect ratios of 9.9, 13.0, and 18.9) were used to ensure a sufficient range of large aspect ratios. For all selected cavity sizes, the ratios of flume width and flow depth within the cavities are smaller than 5, signifying that secondary currents could be induced by the flume sidewalls within the cavities (Nezu and Nakagawa 1993). Therefore, flow measurements were conducted on the centerline of the flume to minimize sidewall

effects. To obtain a fully developed turbulent flow, the flume floor was roughened with uniform sand of median grain size, $d_{50} = 0.36$ mm, throughout the flume length except for the cavity zone. A steady uniform approach flow with a constant depth of 12 cm and average velocity, $U = 26.0$ cm/s, were used in all of the tests. The corresponding Reynolds number ($R_D = UD/\nu$, where $h =$ upstream flow depth and $\nu =$ kinematic viscosity) of the approach flow is about 35,000. The corresponding Reynolds numbers based on cavity depths for the three cavities ($R_D = UD/\nu$) are approximately 15,300, 22,200, and 29,200, respectively. The flow condition was examined before actual experimental works began by using both acoustic Doppler velocimeter and PIV measurements on the center of the flume, 1 m upstream of the cavity leading edge. The detailed experimental setup and calibration procedures can be found in Agarwal (2017).

Fig. 1 shows the PIV measurement sections, central section (CS), and downstream section (DS) within the cavity along the centerline of the flume. The flow fields within the rectangular cavities may be regarded as quasi-two-dimensional. The reference point $(x, z) = (0, 0)$ of the x - z coordinate plane in this study was set at the downstream cavity edge on the flume center (Fig. 1). The PIV system consisted of a 5-W diode-pumped solid-state continuous laser with a wavelength of 532 nm, and a high-speed camera (Phantom Miro 120, Vision Research, Wayne, New Jersey) with 3 GB of memory storage. The flow was seeded with polyamide tracer particles of diameter $= 20 \mu\text{m}$ and specific gravity $= 1.23$ for optimal scattering of the laser light. Using Stoke's law, the settling velocity of the seeding particle was estimated to be $50.1 \mu\text{m/s}$, which is negligible compared with the average approach flow velocity. The high-speed camera operated with a 50-mm focal lens at 150 frames per second. For each case, more than 6,000 images were measured. For the image processing, multiple interrogation windows (starting from 32×32 pixels, then passing with 16×16 pixels and ending with 8×8 pixels; all passes adopted a 50% overlap between adjacent subwindows) were adopted in the correlation algorithm to ensure the accuracy of the velocity calculation. Instead of using the traditional PIV imaging processing method with a fixed time interval Δt between two consecutive images, a multitime interval algorithm method developed by Hsieh et al. (2016) was adopted to reduce the bias error due to the high velocity gradient in the flow field. The multiple time intervals used are $\Delta t (= 6.7 \text{ ms}), 3, 9,$ and $21\Delta t$. This method also includes a digital image high-pass filter technique that was used to reduce reflections and scatterings of the laser light. The PIV data calculation was done with a series of FORTRAN codes, whose detailed procedures can be found in Hsieh (2008) and Hsieh et al. (2016). The maximum error of the instantaneous velocities was found to be about 3.4%. A convergence analysis was also conducted to ensure that the number of images in this study was enough to obtain a stable time-averaged turbulence data output. The experimental conditions are tabulated in Table 1, in which the letters and numerals in the "Measurement section" column

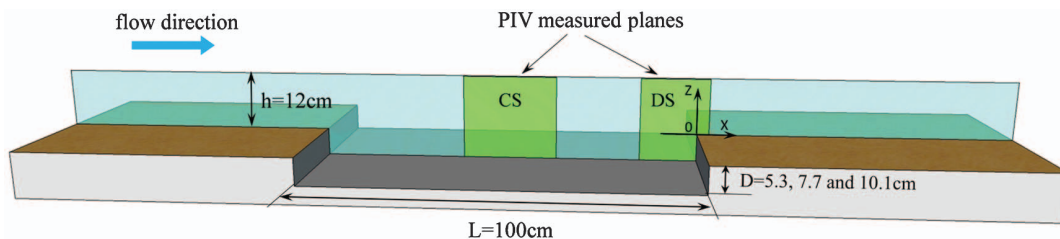


Fig. 1. (Color) Sketch of cavity model and PIV measurement sections.

Table 1. Experimental conditions for PIV measurements

Measurement section	Field of view (cm ²)	Scale (cm/pixel)	Spatial resolution (cm)
T1:1			
T1:2	26.0 × 15.3	0.01854	0.148
T1:3	27.5 × 15.3	0.01886	0.151
T1:4	28.5 × 19.7	0.01978	0.158
T1:5	27.5 × 19.7	0.01961	0.157
T1:6	20.0 × 22.1	0.02110	0.169
T1:7	24.0 × 22.1	0.02109	0.169

(e.g., CS053) refer to the location of the measurement plane (center section) and depth of cavity (53 mm), respectively. The spatial resolution is calculated as the product of the minimum interrogation window (8 pixels) and the measurement scale.

181 Quadrant Analysis Methodology

182 Wallace et al. (1972) conducted the first quadrant analysis of
 183 Reynolds shear stresses in a turbulent shear flow at the wall region
 184 to explore the then-unknown information on turbulence from the
 185 signs of velocity fluctuations. In this study, the quadrant dynamics
 186 are locally affected by the cavity shear layers generated with differ-
 187 ent cavity aspect ratios. The turbulent events are designated accord-
 188 ing to the sign of the fluctuating velocity components. Therefore,
 189 the two-dimensional instantaneous turbulent velocity fluctuations
 190 (u' , w') may be classified into four categories: Q1 ($+u'$, $+w'$),
 191 Q2 ($-u'$, $+w'$), Q3 ($-u'$, $-w'$), and Q4 ($+u'$, $-w'$), which are
 192 customarily called outward interaction, ejection, inward interaction
 193 and sweep events, respectively. As indicated in Pope (2000),
 194 Reynolds shear stress is expressed as $-\langle u'w' \rangle$ and the turbulence
 195 production is calculated as $T_p = -\langle u'w' \rangle (\delta u / \delta z)$; therefore, Q1
 196 and Q3 give negative contributions to the Reynolds shear stress
 197 and lead to energy dissipation, whereas Q2 and Q4 give positive
 198 contributions to the Reynolds shear stress and result in energy pro-
 199 duction. To differentiate the higher magnitude contributions to the
 200 time-averaged Reynolds shear stress $-\overline{u'w'}$ from each quadrant, a
 201 hole-size parameter H is adopted (Nezu and Nakagawa 1993;
 202 Willmarth and Lu 1972). The hyperbolic hole area is determined
 203 by the curve $|u'w'| = H(u'_{rms} w'_{rms})$, in which u'_{rms} and w'_{rms}
 204 are the root mean square of the horizontal and vertical instantaneous
 205 turbulent velocity fluctuations, respectively. With an increase in H ,
 206 the hole area is enlarged and the fractional contribution of the re-
 207 maining data outside of the hole to the total shear stress can be
 208 determined. Thus, the shear stress fraction from each quadrant
 209 can be written as

$$S_{i,H}^f = \langle u'w' \rangle_{i,H} / \overline{u'w'} \quad (1)$$

210 where $\langle u'w' \rangle_{i,H}$ are the contributions from the i th quadrant
 211 to $-\overline{u'w'}$, and is calculated as

$$\langle u'w' \rangle_{i,H} = \sum_{n=1}^N u'(n)w'(n)I_{i,H}(u', w') \quad (2)$$

212 and N = number of total bursting events; n = number of instanta-
 213 neous bursting events; and $I_{i,H}(u', w')$ = detection function,
 214 defined as

$$I_{i,H}(u', w') = \begin{cases} 1, & \text{if } (u', w') \text{ is in quadrant } i \text{ and if } |u'w'| \geq H(u'_{rms}w'_{rms}) \\ 0, & \text{otherwise} \end{cases} \quad (3)$$

The duration fraction $T_{i,H}^f$ for the i th quadrant is defined as 215

$$T_{i,H}^f = \frac{1}{N} \sum_{n=1}^N I_{i,H}(u', w') \quad (4)$$

The conditional statistics of experimental data of the Reynolds 216
 shear stress within the cavity flow are analyzed using these 217
 equations. 218

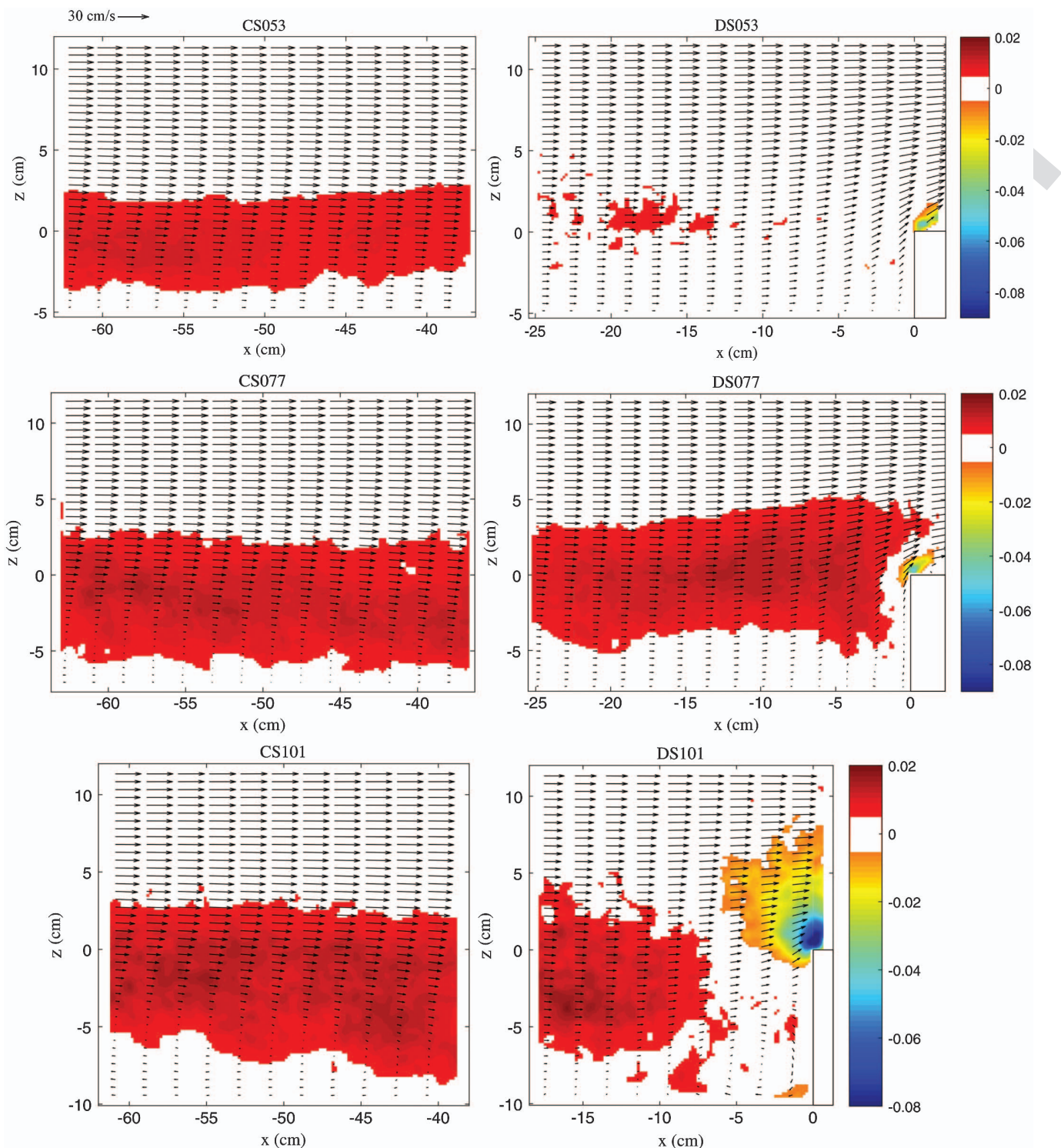
219 Results and Discussion

220 Time-Averaged Velocity and Reynolds Shear Stress 221 Distributions

222 The time-averaged velocity and normalized Reynolds shear stress
 223 ($-u'w'/U^2$) distributions of the measured sections are shown in
 224 Fig. 2. First, the experimental data show that the flow fields of
 225 the regions above $Z = -5$ cm in all three cavities with different
 226 aspect ratios are generally similar. For the largest cavity depth with
 227 $D = 10.1$ cm, an obvious flow recirculation zone can be identified
 228 at the downstream corner of the cavity (see DS101 in Fig. 2).
 229 Second, although only two sections (one at the center and the other
 230 at the downstream edge of the cavity) of the entire cavity flow field
 231 were measured in this study, one still can readily infer from the
 232 experimental data how a well-developed strong positive Reynolds
 233 shear stress layer (indicated by the belt of dark red color in the
 234 regions of -60 cm $< X < -40$ cm, and -3 cm $< Z < 2$ cm of the
 235 CS sections in Fig. 2) that forms at the cavity leading edge has
 236 swept into the cavity and reattached onto the cavity floor. It then
 237 separates before reaching the downstream cavity wall to rejoin the
 238 main flow there. This is consistent with the observed flow patterns
 239 within a large aspect ratio ($L/D = 5.16$) cavity by other research-
 240 ers (Ukeiley and Murray 2005). In compressible cavity flows, such
 241 a flow regime is defined as a *closed* cavity flow (Plentovich et al.
 242 1993); it occurs in very shallow cavities with aspect ratios greater
 243 than 9 (Ashcroft and Zhang 2005). Third, the values of the normal-
 244 ized time-averaged Reynolds shear stress of all the cases are very
 245 small and fluctuate around zero above the strong positive shear
 246 stress layer. This is due to the unsteady small vertical velocity gra-
 247 dients. Fourth, Fig. 2 also reveals that a strong negative Reynolds
 248 shear stress zone is formed immediately downstream of the cavity
 249 downstream edge due to contraction of the flow cross section
 250 for all three cases. Moreover, the time-averaged flow and turbu-
 251 lence structure at the cavity downstream edge are found to be sim-
 252 ilar to those on a forward-facing step (Ren and Wu 2011). This is
 253 because, for a cavity with large aspect ratios, the flow can reattach
 254 onto the bottom of the cavity and redevelop to a quasi-uniform state
 255 before reaching the downstream cavity edge. However, to deter-
 256 mine a critical aspect ratio that renders cavity flows to resemble
 257 that with a forward-facing step still needs further study. Finally,
 258 the increase of cavity depth increases the magnitude of the positive
 259 Reynolds shear stress layer within the cavity and the negative
 260 Reynolds stress zone immediately upstream of the cavity down-
 261 stream edge.

262 Velocity Fluctuations

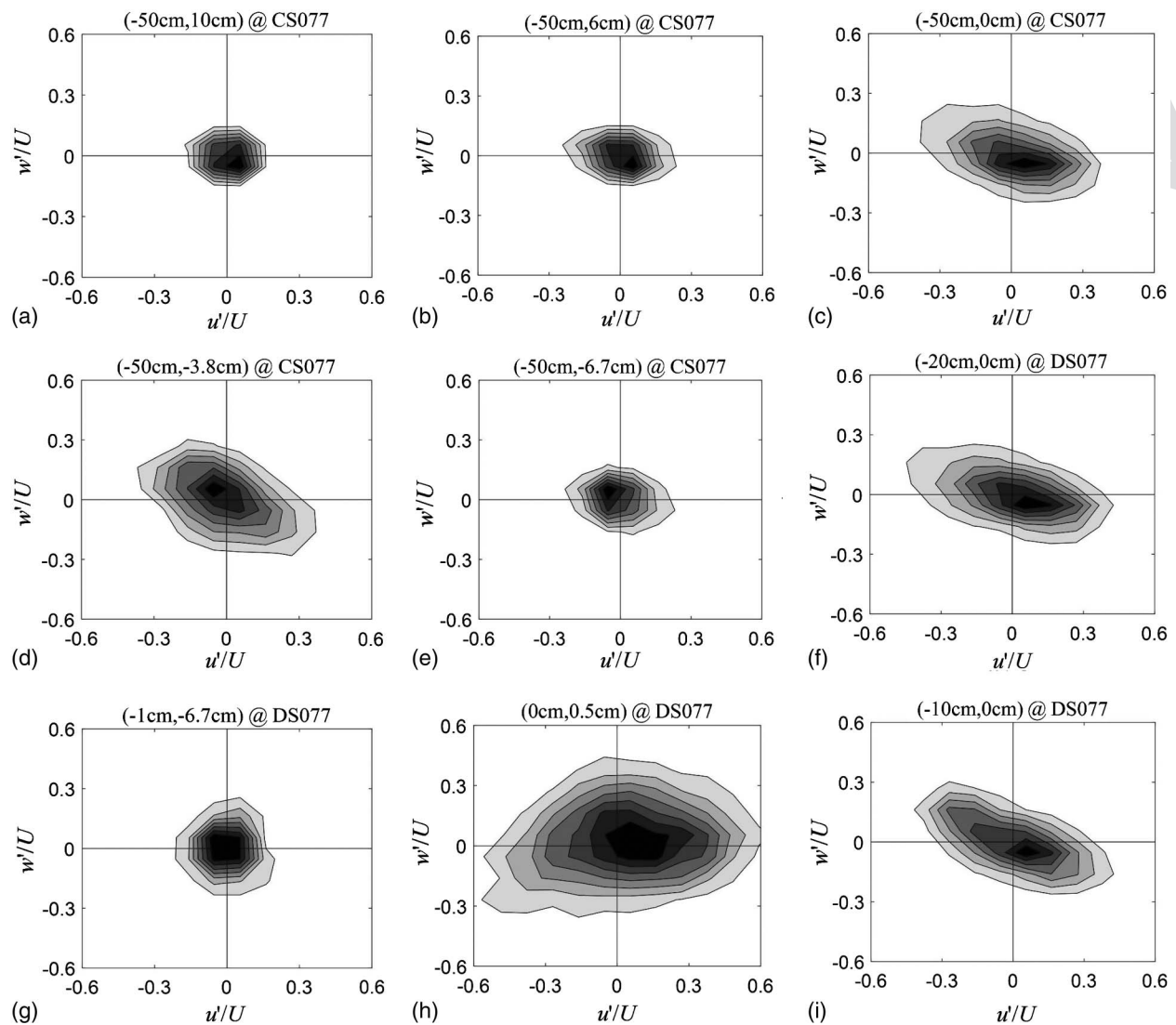
263 Fig. 3 shows quadrant maps of the fluctuating component of veloc-
 264 ity u' and w' at selected locations for Tests CS077 and DS077.
 265 Instead of a direct presentation of scatter plots, the data have been
 266 contoured in terms of probability density and normalized by the
 267 cavity upstream approach average velocity for a better comparison.
 268 Therefore, the plots in Fig. 3 are joint probability distribution
 269 functions (PDFs) of the two fluctuating components of velocities



F2:1 **Fig. 2.** (Color) Time-averaged velocity vectors and normalized Reynolds shear stress distributions (the color scale represents the magnitude of
 F2:2 normalized Reynolds shear stress $-u'w'/U^2$; flows from left to right).

270 u' and w' . The joint probability of the paired velocity fluctuations
 271 $P(u', w')$ in a given region in the $u'-w'$ plane can be calculated
 272 based on the occurrence numbers of the instantaneous velocity fluctuations
 273 over the region. The darkest zones in each plot correspond
 274 to the largest concentrations of instantaneous fluctuations for that
 275 position. Moreover, the sizes and shapes of the joint PDFs at differ-
 276 ent locations indicate different turbulence properties. A careful
 277 examination of the data at locations [Figs. 3(c, f, and i) on the right
 278 column of the figure] along the cavity top ($Z = 0$ cm) reveals that

the velocity fluctuations have a similar turbulence intensity and
 appear as an elliptical shape, with their major axis skewed into the
 Q2 and Q4 quadrants. This is consistent with the strong positive
 Reynolds shear stresses and the most frequent ejections and sweep
 events that form at these locations (Fig. 2). On the cavity center and
 above the cavity top, however, the major axis of the elliptical joint
 PDF reduces its length and rotates anticlockwise as Z increases
 [Figs. 3(a-c), top row], indicating a reduced turbulent-like behavior.
 On the cavity center and within the cavity ($Z < 0$), strong



F3:1 **Fig. 3.** Joint PDFs of normalized velocity fluctuations u' and w' at specific locations for Tests CS077 and DS077 (coordinates of the locations are
F3:2 shown on the top of each subplot).

288 velocity fluctuations still can be observed within the band of positive
289 Reynolds shear stress layer [Fig. 3(d)]. The small joint PDF of
290 Fig. 3(e) (1 cm above the center of the cavity floor) shows that the
291 turbulence is restricted near the cavity floor. At the downstream
292 corner (-1 cm, -6.7 cm) of the cavity [Fig. 3(g)], a symmetrical
293 distribution (circular) of joint PDF is observed. This indicates
294 that the turbulence is small and nearly isotropic at this location.
295 Immediately above the downstream cavity edge [Fig. 3(h)], the
296 distribution of velocity fluctuations also adopts an elliptical shape,
297 but with its major axis skewed into the Q1 and Q3 quadrants. The largest
298 size of PDF (of all the subplots) at this location indicates its
299 strongest turbulence intensity in the measured planes. This is a
300 reflection of the strongest negative Reynolds shear stress developed
301 at this location (Fig. 2).

302 Dominant Quadrant Event Distributions

303 For a specific hole-size H , the dominant quadrant of the highest
304 probability of occurrence at a given location is obtained by comparing
305 the magnitude of the duration fractions $T_{i,H}^f$. The dominant
306 quadrant event distributions of the measured sections are shown in
307 Fig. 4. Two hole sizes, $H = 0$ and 1, are used in this part of the

analysis. The events associated with Q2 and Q4 are the most
308 frequent in the measured flow field for all three cases. For $H = 0$,
309 the results associated with the three center sections (CS053,
310 CS077, and CS101) show a similar pattern of dominant quadrant
311 distribution within and below the cavity shear layer (indicated as
312 the strong positive Reynolds shear stress layers in Fig. 2; see
313 the regions of $-60 \text{ cm} < x < -40 \text{ cm}$, and $-3 \text{ cm} < Z < 2 \text{ cm}$).
314 Specifically, Q4 has the highest probability of occurrence within
315 the zone of the shear layer, and Q2 almost dominates all areas
316 within the cavity and a certain depth of flow immediately above
317 the shear layer. This feature is consistent with previous works
318 in stepped cavity flows (e.g., Amador et al. 2006). For the flow
319 layers above the cavity shear layer, the dominant quadrant
320 distributions are different for the three different cases. As mentioned
321 in the previous section, the reattached position of the shear
322 layer is strongly dependent on the cavity aspect ratios. Therefore,
323 it can be inferred that (1) the downstream shift and progressive
324 enlargement of the Q1 region associated with the small negative
325 time-averaged Reynolds shear stresses (Fig. 2), and (2) the reducing
326 dominance of the Q2 region above the cavity shear layer in
327 CS077 and CS101, are closely related to the decrease of the cavity
328 aspect ratio.
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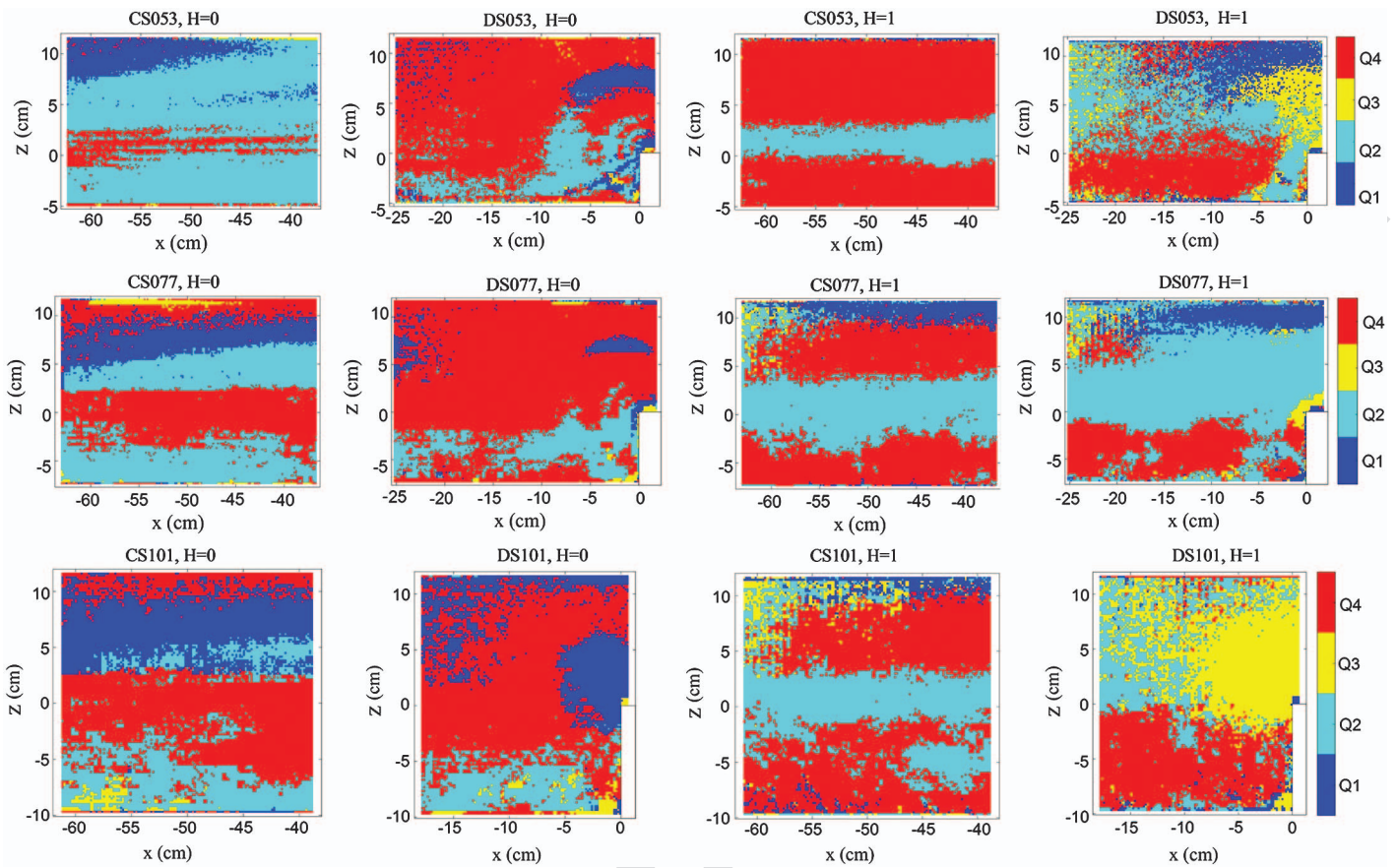


Fig. 4. (Color) Dominant quadrant event distribution of measured sections.

F4:1

For $H = 0$, Q4 dominates almost all flow areas above the cavity, and Q2 is the most frequent event within the cavity in all three downstream sections (DS053, DS077, and DS101). Q1 and Q3 have higher occurrences at the cavity corners and around the cavity downstream edge, which corresponds to the areas with negative Reynolds shear stresses. For the high-magnitude events ($H = 1$), quadrant dominances are almost inverted for all measured planes. This feature, which is consistent with the findings in Amador et al. (2006), indicates an asymmetry of high infrequent velocity fluctuations. Although these patterns of quadrant maps in Fig. 4 appear to be clear, an erroneous interpretation is possible because of a slight dominance from specific events (Buffin-Bélangier and Roy 1998).

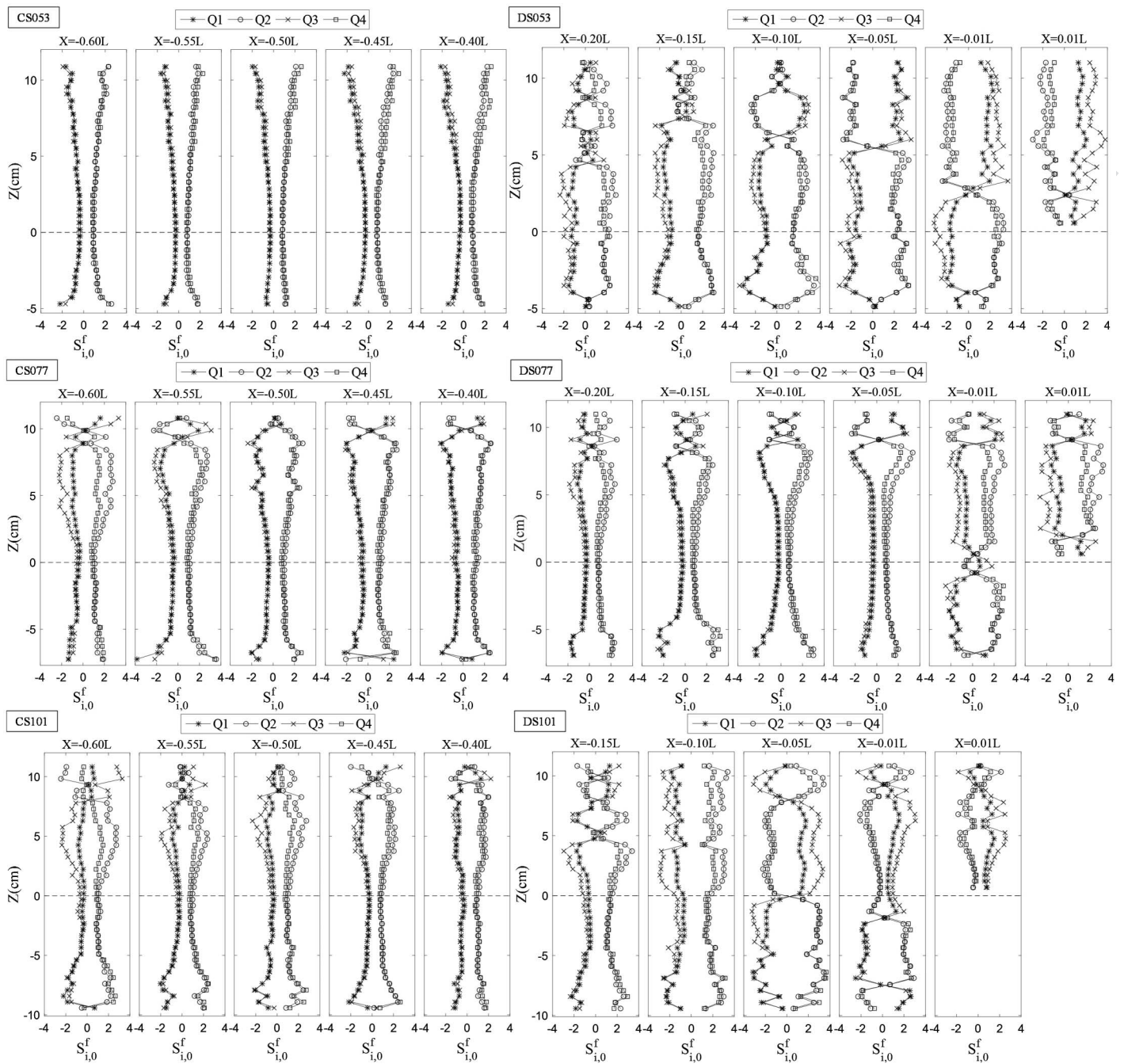
Quadrant Shear Fractions

Fig. 5 shows the profiles of fractional Reynolds shear stress contributions, $S_{i,H}^f$, for $H = 0$ at different locations along the measured sections. Clearly, the cavity aspect ratio has a significant effect on the distributions of the quadrant shear fractions because of its influence in modifying the flow structures within the cavity. Although the distributions of shear fractions for all three cases are not identical, some common properties still can be deduced from Fig. 5. For the regions having high absolute time-averaged Reynolds shear stress values, such as regions of the cavity shear layer and around the cavity downstream edge, the absolute values of contribution fractions for all quadrants are within a small range from 0 to 1.5. These small Reynolds shear stress fractions provide an interesting feature, indicating that the opposing events are approximately balanced. For the rest of the regions having small absolute time-averaged Reynolds shear stress values, the absolute

values of contribution fractions are within a relatively larger range from 1.5 to 4. In these regions, Q2 and Q3 are found to be the dominant contributions among the opposing events. A cursory comparison with Fig. 3 reveals that these regions have the highest occurrence probability of Q1 and Q4. These contrasting observations indicate that a larger proportion of the shear stress is generated over shorter durations. In a previous study on transition boundary layers, similar observations were found by Nolan et al. (2010).

To further investigate the high-magnitude quadrant events on the cavity shear layers, the fractional Reynolds shear stress contributions against hole size H at three locations $[(-50 \text{ cm}, 0 \text{ cm}), (-10 \text{ cm}, 0 \text{ cm}), \text{ and } (0 \text{ cm}, 0.5 \text{ cm})]$ are plotted in Fig. 6. The dash curve T in the plots represents the percentage of time that the data are outside the hole with a certain value of H . For all locations in the three cases and $H = 2$, Fig. 6 shows that the motions outside of the hole occur less than 5% of the total time, whereas the total Reynolds shear stress results from all these motions still account for considerable proportions. This behavior indicates that the high intense, intermittent motions are the major source of Reynolds stress. For the two locations within the cavities, Q2 and Q4 are the dominant positive contributions to the total Reynolds shear stresses, whereas for the locations on the cavity's downstream edge, Q1 and Q3 are the dominant positive contributions. This is consistent with the signs of the total Reynolds shear stresses at these locations observed in Fig. 2. The balancing feature of the opposing events at locations with high absolute time-averaged Reynolds shear stresses is sustained and asymptote to a constant value of zero as the hole size increases [Figs. 6(a, c, e, f, and i)]. This feature indicates an evenly distributed elliptical or circular shape of velocity fluctuations on the quadrant map. Figs. 6(c, f, and i) also indicate that, for the

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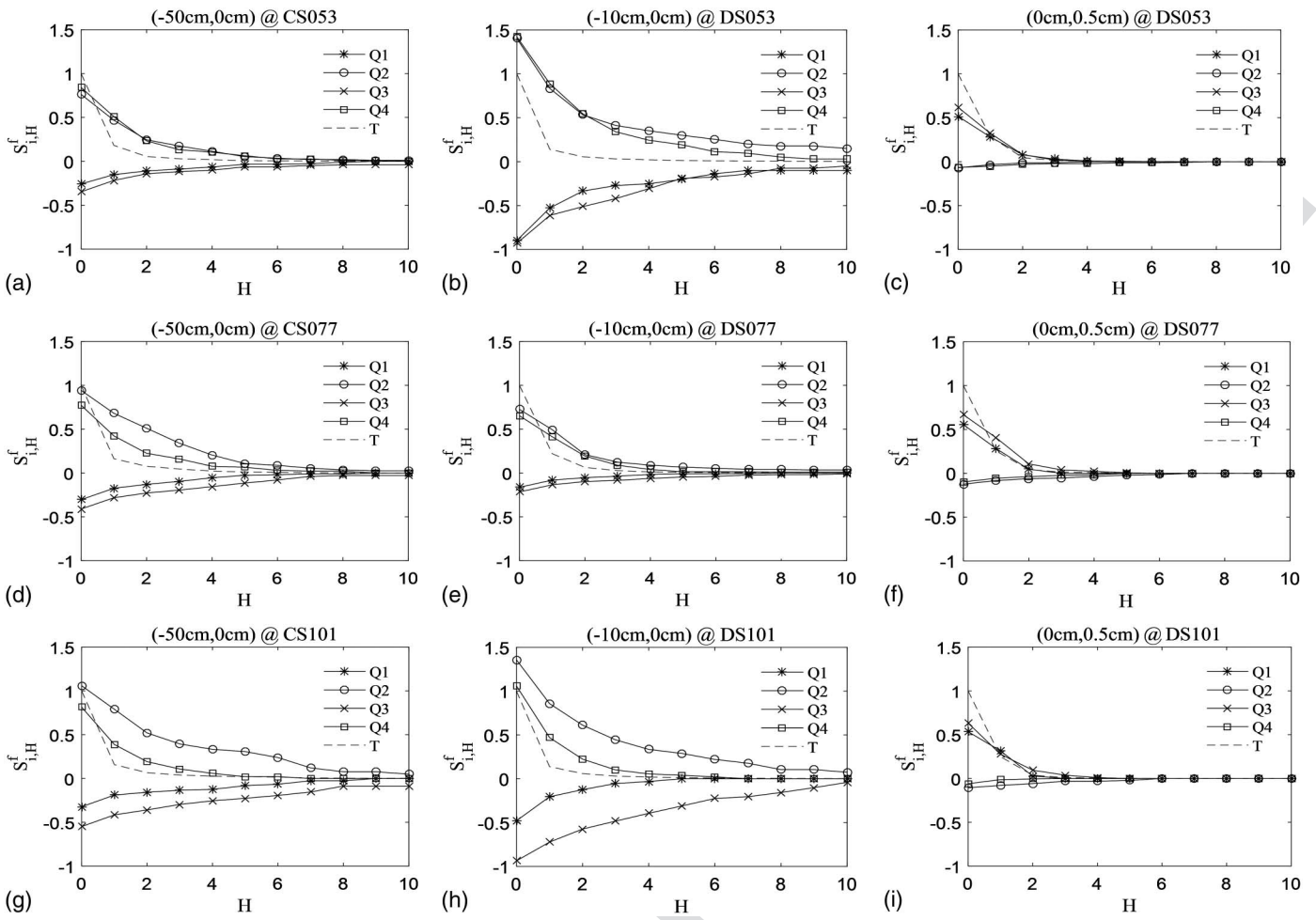
F5:1 **Fig. 5.** Vertical profiles of quadrant shear fractions, $S_{i,0}^f$, along the measured planes for the three cavities (locations of vertical profiles are shown on
 F5:2 top of each subplot).

388 locations with larger Reynolds shear stress, all contributions from
 389 the four quadrants vanish quicker with an increase of the hole size.

390 Conclusions

391 This paper presents an experimental study on the quadrant analysis
 392 of turbulence structures in a rectangular cavity with comparatively
 393 large aspect ratios using a high spatial resolution PIV technique.
 394 New insights on the turbulence structures of the time-averaged flow
 395 within the cavity of large aspect ratios are presented. The measured
 396 time-averaged velocity field and Reynolds shear stress distributions
 397 within the cavity show that the flow pattern and turbulence struc-
 398 tures are strongly affected by the cavity aspect ratio. With an

increase of the cavity depth, the negative Reynolds shear stress re- 399
 gion at the cavity's downstream edge increases its intensity in both 400
 magnitude and extent. For $H = 0$, the dominant quadrant event 401
 distributions of all three cavity sizes show a similar feature 402
 (i.e., Q4 has the highest probability of occurrence within the zone 403
 of the shear layer, and Q2 dominates almost all areas within the 404
 cavity and a certain depth of flow immediately above the shear 405
 layer). For $H = 1$, quadrant dominances in all cavity sizes are in- 406
 verted. The quadrant shear stress distributions show that the oppos- 407
 ing events are approximately balanced on the cavity shear layers 408
 and near the cavity's downstream edge. This balanced feature in- 409
 dicates an evenly distributed elliptical or circular shape of velocity 410
 fluctuations on the quadrant map. The quadrant hole analysis for 411
 locations on the cavity shear layers reveals that the percentage of 412



F6:1 **Fig. 6.** Fractional contributions to Reynolds shear stress, $S_{i,H}^f$, at noted locations as a function of H (coordinates of the locations are shown on top
F6:2 of each subplot; T represents the percentage of time that the data are outside of the hole, and uses the same vertical scale as $S_{i,H}^f$).

413 time that the motions are outside of the hole drops much quicker
414 than those of the shear fractions. This indicates that small periods of
415 strong velocity fluctuations contribute large proportions to the total
416 Reynolds shear stress at these locations. Finally, the present exper-
417 imental results of this study will be useful in the turbulence mod-
418 eling of cavity flows.

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427 Notation

428 *The following symbols are used in this paper:*

- 429 D = cavity depth (m);
430 d_{50} = median sediment size (m);
431 H = hole size;
432 h = approach flow depth (m);
433 $I_{i,H}$ = detection function;

- i = quadrant numbers (i.e., 1, 2, 3, 4); 434
 L = cavity length (m); 435
 N = number of total bursting events; 436
 n = number of instantaneous bursting events; 437
Q1 = outward interaction events; 438
Q2 = ejection events; 439
Q3 = inward interaction events; 440
Q4 = sweep events; 441
 R_D = Reynolds number based on cavity depth; 442
 R_e = Reynolds number of the approach flow; 443
 $S_{i,H}^f$ = shear stress fraction for the i th quadrant within the 444
hole size H ; 445
 T = percentage of time that the data are outside of the 446
hole; 447
 $T_{i,H}^f$ = duration fraction for the i th quadrant within the hole 448
size H ; 449
 T_P = energy production (m^2/s^3); 450
 U = upstream average velocity (m/s); 451
 u' = horizontal instantaneous turbulent velocity 452
fluctuations (m/s); 453
 u'_{rms} = root mean square of horizontal instantaneous 454
turbulent velocity fluctuations (m/s); 455
 ν = viscosity of water (m^2/s); 456
 w' = vertical instantaneous turbulent velocity fluctuations 457
(m/s); 458

459 w'_{rms} = root mean square of vertical instantaneous turbulent
 460 velocity fluctuations (m/s);
 461 Δt = time interval between the two consecutive images (s);
 462 $-\langle u'w' \rangle$ = instantaneous Reynolds shear stress (m^2/s^2);
 463 $-\overline{u'w'}$ = time-averaged Reynolds shear stress (m^2/s^2); and
 464 $\langle u'w' \rangle_{i,H}$ = contributions from the i th quadrant to $-\overline{u'w'}$
 465 (m^2/s^2).

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